

Comsol Laboration: Heat Conduction in a Chip

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A chip on a circuit board is heated inside and cooled by convection by the surrounding fluid.

- What is the steady state temperature distribution, and how long does it take to reach steady state ?
- How do the chip dimensions influence the max. temperature?

1 Model

The mathematical model is an initial-boundary value problem for heat equation, to determine $T(x, y, z, t)$ for $t > 0$ when

$$\rho C \frac{\partial T}{\partial t} = \nabla \cdot (k \nabla T) + Q \text{ in } \Omega$$

$$k \frac{\partial T}{\partial n} = h (T_{ext} - T) \text{ on } \partial\Omega$$

with initial data $T(x, y, z, 0) = f(x, y, z)$

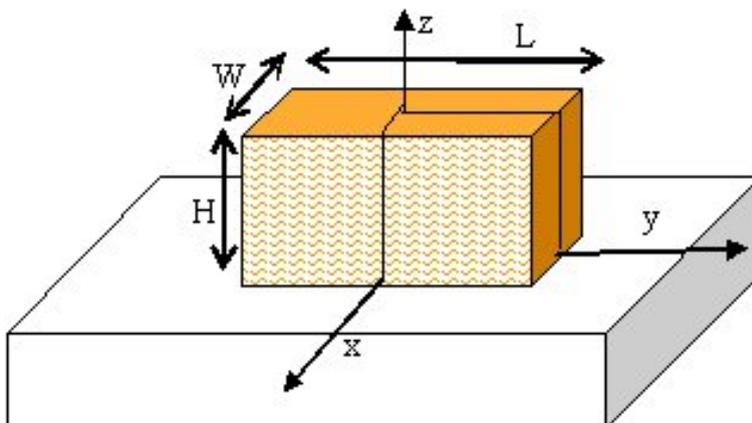


Figure 1: Chip geometry

ρ is the mass density [kg/m^3], specific heat C [$\text{J}/(\text{kg K})$], k thermal conductivity [$\text{W}/(\text{mK})$] and Q the heat source [W/m^3]. T_{ext} is ambient temperature and h the convection heat transfer coefficient [$\text{W}/(\text{m}^2\text{K})$] between the chip surface and surroundings.

We make some simplifying assumptions:

1. No heat transfer to the card
2. Constant heat transfer coefficient h on all the other sides.
3. Uniform heat source Q
4. The chip is thin and long, $H \ll W \ll L$, and we shall study the impact of thinness below. In the future, we use symmetry to compute on a $H \times (W/2) \times (L/2)$ part of the chip.

Data for silicon: $\rho = 2300[\text{kg}/\text{m}^3]$, $C = 380[\text{J}/\text{KgK}]$, $k = 3.6[\text{W}/\text{mK}]$. Total power is P [W], ie. $Q = \frac{P}{V}[\text{W}/\text{m}^3]$ where V is the volume of the chip. The heat transfer coefficient h when the chip is cooled by a weak fan is h_{coef} (E.g. 100) $\text{W}/\text{m}^2\text{K}$ and $T_{ext} = 20^\circ\text{C}$.

2 Preliminary analysis

- a) Symmetry makes it sufficient to simulate 1 / 4th of the chip.

Proof: Suppose that the solution is unique, say $T(x, y, z, t)$. Show that $T(-x, y, z, t)$ and $T(x, -y, z, t)$ are also solutions.

- b) An important characteristic quantity is the *Biot number* $Bi = \frac{h}{k/d}$, the ratio of thermal resistance $1/h$ for convection and d/k for the chip itself, where d is a characteristic length for heat transport through the chip. Here H is the smallest dimension. When Bi is large, the surface temperature is close to T_{ext} , and when Bi is small, the temperature will be nearly uniform in the body.
- c) What is the temperature when Bi (ie h) is small?
- d) Assume that the chip, after being on for a long time, is turned off at $t = 0$ so $Q = 0$ for $t > 0$. The Newton cooling law, often used in crime fiction to determine time of death, is

$$\tau \frac{dT}{dt} = T_{ext} - T$$

which is the limiting case of vanishing Bi .

- e) What is the time-scale τ for the chip? Tip: Start from the PDE 1, integrate over the entire chip and use Gauss's theorem. Use that T is almost constant through the chip.
- f) Derive a similar model for heating when the chip is turned back on.

3 In the Lab-session

- **Model:** Use 3D, Classical PDES / Heat Equation. It is worthwhile to define the material parameters as `Constant` under `Options`. Then you can easily change values etc, and the equations get to be in understandable terms.
- **Geometry:** A brick ! give the right dimensions under `properties`
- **PDE:** Set the value of k and Q (named `f` in the GUI menu)
- **Boundary conditions:** Neumann-conditions : insulation on the bottom and symmetry-boundaries is obtained by `q = g = 0`. On the cooled surfaces, with `q` and source term `g` with h_{coef} and $h_{coef} \times T_{ext}$ these may first be defined as the `Constant` under `Options`.
- **Solver :** `Stationary linear`, except for transient runs (e) conducted with `Time Dependent`. Choose a finish time based on the calculated time-scale.

1. (A) Calculate the maximum temperature when

$$P = \dots \text{ W } \dots \text{ yy}$$

$$h_{coef} = 20 \text{ W /m K } \dots \text{ dd}$$

$W = 5 + \text{mm} / 2 \dots \text{mm}$

where the analyst's birth date is yy mm dd.

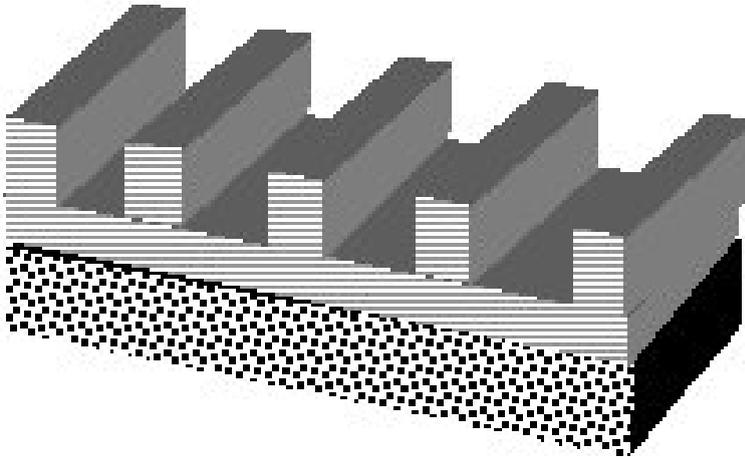


Figure 2: Chip with cooling fins

2. (B) for $W = 10 \text{ mm}$, do the nätförfinings study and try the P2 and P1 - elements.
3. (C) Plot the temperature profiles and find maximum temperatures for $W = 10.1, 0.2 \text{ mm}$
4. (D) Derive for $h = \infty$ the approximate value $T_{max} = H^2 Q / (2k)$. Is this perhaps an upper or lower limits? Tip: The differential equation can be simplified to

$$0 = k \frac{d^2 T}{dz^2} + Q, \frac{dT}{dz} \Big|_{z=0} = 0, T(h) = T_{ext}$$

5. (E) The question "Is it as fast to heat up as to cool down?" is imprecise. Please specify, observe and justify/ motivate . It is easiest to run a simulation starting with $T = T_{ext}$, turn on Q at $t = 0$ and shut off after so long time that the temperature has become (almost) stationary: $Q = (t < t_{end}) \times Q_0$
6. (F) The temperature can be reduced by cooling. Let the chip-octant be $10 \times 10 \times 2 \text{ mm}^3$. Place a 1 mm thick aluminum cap on the chip, then 5 pcs. 1 mm thick and 1 cm high cooling fins:

Data for aluminum: $\rho = 1600 \text{ [kg/m}^3\text{]}$, $C = 380 \text{ [J / (kg K)]}$, $k = 60 \text{ [W / (m K)]}$

How much is the maximum temperature reduced by the cooling fins?

Is it worthwhile to make the flanges higher and / or thinner and more numerous?

A thought: Total surface area exposed to air is an important parameter in the model (see (a) above). Then, maybe we could do with just a little Al, by making many densely packed thin flanges. But that is not how the actual chip cooler looks . Why? What is wrong with our model? **Geometry:** First one creates the top as a brick and places it by the filling the right geometry data under Draw /Object Properties:

Then make an outer flange, place it, then copy it and give the right translation in the dialog box (four times). Create the Composite Object from the union of all the flanges and the lid

without interior borders. Finally, make an object composed of top, flanges, and chip, with retained interior borders. That's it!

PDE: Fill materials data and initial data in the two Subdomains (Al and Si).

Boundary conditions: Select Neumann for *all* boundaries, with heat transfer coefficient q and source term g like h_{coef} and $h_{coef} * T_{ext}$ - these may first be defined as the Constant under Options. Then change the (few) symmetry-boundaries to insulation.